

The 'velocity of shrinkage' for points on the particle surface is easily obtained as

$$\frac{ds_s}{dt} = \frac{M}{\rho} N_{us} = \frac{M}{\rho} k_c C_{\infty} \quad (12)$$

Where  $ds_s$  is the element of length perpendicular to the particle surface and this is constant over the surface of the particle. As a result, the ratio  $b/a$  between minor and major axes of the ellipse decreases in the process of reaction and so the burning particle deviates increasingly from a sphere as the reaction proceeds. Indeed, in any period of time,  $b_i$  and  $a_i$  are reduced by the same amount, say  $s$ , and for  $b_i < a_i$  it is obvious that  $(b_i/a_i) > (b_i - s)/(a_i - s)$ .

The evolution of particle shape may be easily obtained from the original ellipsoid by 'peeling off' layers of constant thickness.

*Int. J. Heat Mass Transfer.* Vol. 29, No. 10, pp. 1607-1609, 1986  
Printed in Great Britain

0017-9310/86 \$3.00 + 0.00  
Pergamon Journals Ltd.

REFERENCES

1. M. R. Spiegel, *Schaum's Outline of Theory and Problems of Vector Analysis and an Introduction to Tensor Analysis*. Schaum, New York (1959).
2. R. P. Pecanha and B. M. Gibbs, The importance of coal fragmentation and swelling on coal burning rates in a fluidised bed combustor. In *Fluidised Combustion: Is it Achieving its Promise?*, DIS/9/65-71. Institute of Energy, London (1984).
3. I. B. Ross and J. F. Davidson, The combustion of carbon particles in a fluidised bed, *Trans. Inst. chem. Engrs* **59**, 108-114 (1981).
4. C. M. C. T. Pinho and J. R. F. Guedes de Carvalho, The combustion of coke particles in a fluidised bed—some aspects of kinetic data collection, *8th Int. Symposium on Chemical Reaction Engineering*, pp. 77-84, Edinburgh (1984).

Average Nusselt number on the downward-facing heated plate

D. K. EDWARDS and J. C. HALAD

Department of Mechanical Engineering, University of California—Irvine,  
Irvine, CA 92717, U.S.A.

(Received 26 November 1985 and in final form 25 February 1986)

INTRODUCTION

IN A RECENT paper Schulenberg [1] determined analytically the stagnation point heat transfer coefficient for natural convection on the horizontal downward-facing heated plate. In his Fig. 9 he plotted 12 points based upon experimental data for the ratio of average to stagnation point Nusselt number. Hatfield and Edwards [2] had correlated average Nusselt number on the basis of a virtual extension,  $x_0$ , of the boundary layer, as shown in Fig. 1, to allow for the finite thickness of the boundary layer as it flowed around the corner of the plate, or outwards onto a horizontal adiabatic extension of length  $L_a$ . In this note we apply the virtual displacement concept to find a closed-form correlation based upon Schulenberg's analysis and compare it to the previous correlation [2].

ANALYTICAL BASIS

Given a local Nusselt number  $Nu_x$  that varies with local  $Ra_x$ , where  $x$  is measured from the virtual edge, the average  $Nu_L$  based upon total length  $L$  is

$$Nu_L = \frac{\bar{h}L}{k} = \frac{L}{k} \frac{2}{L} \int_{x_0}^{x_0+L/2} \left( \frac{k}{x} Nu_x \right) dx \quad (1)$$

In keeping with the approximate boundary-layer theory of Singh *et al.* [3] and the stagnation point solution of Schulenberg [1], a one-fifth power relationship is assumed

$$Nu_x = C'(Pr) Ra_x^{1/5} \quad (2)$$

The result of Schulenberg for the stagnation point can be expressed as a Nusselt number based upon total length  $L$

$$Nu_0 = \frac{h_0 L}{k} = 2^{2/5} C(Pr) Ra_L^{1/5} \quad (3)$$

where for the infinite isothermal strip Schulenberg gives

$$C(Pr) = \frac{0.571 Pr^{1/5}}{(1 + 1.156 Pr^{3/5})^{1/3}} \quad (4)$$

Coefficient  $C'(Pr)$  in equation (2) is related to  $C(Pr)$  in equation (3) by equating  $h_0$  to the local heat transfer coefficient at  $x_0 + L/2$ . Equation (2) may then be substituted into equation (1) and the integration carried out. The result is

$$\frac{\bar{h}}{h_0} = \frac{Nu_L}{Nu_0} = \frac{5}{3} (1 + \zeta)^{2/5} [(1 + \zeta)^{3/5} - \zeta^{3/5}] \quad (5)$$

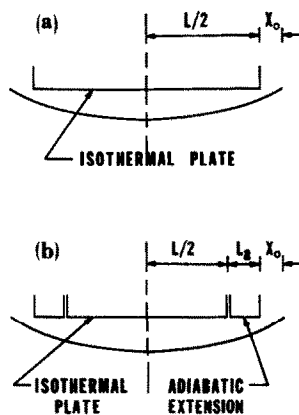


FIG. 1. Schematic of the boundary layer on a heated horizontal plate facing down. (a) Bare edges; (b) adiabatic extensions.

## NOMENCLATURE

$c$	specific heat capacity	$Ra_L$	Rayleigh number, $g\beta TL^3\nu^{-1}\alpha^{-1}$
$C(Pr)$	dimensionless parameter defined by equation (4)	$Ra_x$	local Rayleigh number, $g\beta Tx^3\nu^{-1}\alpha^{-1}$
$C'(Pr)$	dimensionless parameter in equation (2)	$T$	temperature
$C_1, C_2, C_3, C_4$	empirical constants in ref. [2]	$W$	long side length of plate
$\bar{h}$	average heat transfer coefficient defined by equation (1)	$x_0$	edge displacement length (virtual extension).
$h$	local heat transfer coefficient	Greek symbols	
$h_0$	heat transfer coefficient at the stagnation point	$\alpha$	thermal diffusivity, $k/\rho c$
$k$	thermal conductivity	$\beta$	coefficient of thermal expansion, $-(1/\rho)\partial\rho/\partial T$
$L$	short side length of plate	$\zeta$	dimensionless parameter in equation (4), $2x_0/L$
$L_a$	length of adiabatic extension	$\Delta T$	wall to free-stream temperature difference
$Nu_L$	average Nusselt number, $\bar{h}L/k$	$\nu$	kinematic viscosity
$Nu_x$	local Nusselt number, $hx/k$	$\rho$	fluid density.
$Nu_0$	Nusselt number at the stagnation point based on $L$ , $h_0L/k$		
$Pr$	Prandtl number, $\nu/\alpha$		

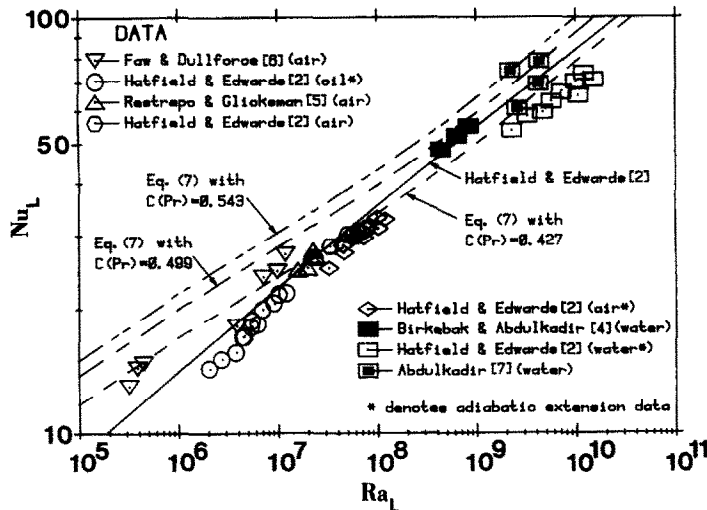


FIG. 2. Comparison of correlations with experimental data (the asterisk denotes data with adiabatic extensions plotted on a length basis of  $L+2L_a$ ).

where  $\zeta = 2x_0/L$ . In the limit of small  $\zeta$  the average to stagnation point ratio goes to  $(5/3)(1-\zeta^{3/5})$ . In the limit of large  $\zeta$  it goes to  $[1+(5\zeta)^{-1}]$ .

## CORRELATION

Equation (5) permits Schulenberg's  $\bar{h}/h_0$  vs  $Ra$  figure to be recast as  $\zeta$  vs  $Ra_L$  where  $L$  is again based upon the total length. Disregarding the low Prandtl number point, we choose to fit the data with

$$\zeta = 2.87 \times 10^{-5} Ra_L^{3/5} \quad (6)$$

The  $(1+0.38L/W)$  aspect ratio correction of ref. [2] is retained where  $W \geq L$  is the long dimension of a rectangle. Accordingly the expression for  $Nu_L$  can be written as

$$Nu_L = 2^{2/5} C(Pr) Ra_L^{1/5} (1+0.38L/W) (\bar{h}/h_0) \quad (7)$$

where  $\bar{h}/h_0$  is given by the RHS of equation (5) with  $\zeta$  given by equation (6). The correlation, equation (7), is thus made with only the two arbitrary constants in equation (6).

Figure 2 shows data from refs. [2, 4-7] compared to equ-

ation (7). Data points are shown corrected to squares. Data with small adiabatic extensions are plotted as  $Nu$  based upon  $L+2L_a$  vs Rayleigh number similarly based. Shown also is the correlation equation (12) of ref. [2] for  $L_a/L = 0$ . Note that the correlation constants in ref. [2] were scrambled; the correctly ordered values are  $C_1 = 6.5$ ,  $C_2 = 13.5$ ,  $C_3 = 2.2$  and  $C_4 = 0.38$ . The values of  $C(Pr) = 0.427$ ,  $0.499$  and  $0.543$  in Fig. 2 correspond to  $Pr = 0.7$ ,  $6$ , and  $4800$ , respectively; thus air, water and oil.

The new correlation, equation (7) here, differs from the old correlation, equation (12) in ref. [2], in three respects. First, the old correlation has no  $Pr$  dependency except through Rayleigh number. Second, it indicates lower values of  $Nu_L$  at low values of  $Ra_L$ . Third, the old correlation made specific provision for adiabatic extensions with two additional empirical constants. The present correlation makes no specific provision, and the data with nonzero  $L_a$  plot somewhat low.

The data in the figure do not indicate unambiguously whether the strong  $Pr$  dependency in equation (7) really exists. Data for oil seem to have been reported only for  $3:1$

rectangles with adiabatically extended edges [2]. These data, plotted as  $Nu$  based upon  $L + 2L_a$  vs  $Ra$  similarly based, plot 45.6% lower than the new correlation, but only 23.9% lower than the old correlation, in the vicinity of  $Ra_L = 5 \times 10^6$ . The new correlation agrees well with Birkebak and Abdulkadir's data [4, 7] in the vicinity of  $Ra_L = 10^9$  and  $Ra_L = 4 \times 10^9$ , respectively. The agreement is also good with the air data of Restrepo and Glicksman [5] and Faw and Dullforce [6] for  $Pr = 0.7$ , while the old correlation underpredicts the data of Faw and Dullforce at  $Ra_L = 4 \times 10^5$ .

#### REFERENCES

1. T. Schulenberg, Natural convection heat transfer below downward facing horizontal surfaces, *Int. J. Heat Mass Transfer* **28**, 467-477 (1985).
2. D. W. Hatfield and D. K. Edwards, Edge and aspect ratio effects on natural convection from the horizontal heated plate facing downwards, *Int. J. Heat Mass Transfer* **24**, 1019-1024 (1981).
3. S. N. Singh, R. C. Birkebak and R. M. Drake, Jr., Laminar free convection heat transfer from downward-facing horizontal surfaces of finite dimensions. In *Progress in Heat and Mass Transfer* (edited by T. F. Irvine, Jr., W. E. Ibele, J. P. Hartnett and R. J. Goldstein), Vol. 2, pp. 87-98. Pergamon Press, London (1969).
4. R. C. Birkebak and A. Abdulkadir, Heat transfer by natural convection from the lower side of finite horizontal heated surface, 4th Int. Heat Transfer Conference, Paris, Vol. IV, paper NC2.2 (1970).
5. F. Restrepo and L. R. Glicksman, The effect of edge conditions on natural convection from a horizontal plate, *Int. J. Heat Mass Transfer* **17**, 135-142 (1974).
6. R. E. Faw and T. A. Dullforce, Holographic interferometry measurement of convective heat transport beneath a heated horizontal plate in air, *Int. J. Heat Mass Transfer* **24**, 859-869 (1981).
7. A. Abdulkadir, Flow studies and heat transfer from lower side of finite horizontal heated surface. M.S. thesis, University of Kentucky (1968).